

# 2D axial-symmetric model for fluid flow and heat transfer in the melting and resolidification of a vertical cylinder

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Direct Metal Laser Deposition (DMLD) is an original technique from rapid prototyping, part repairing and surface treatment of metals. This process involves injecting metal powder through a coaxial nozzle into a melt pool obtained by a moving laser beam.

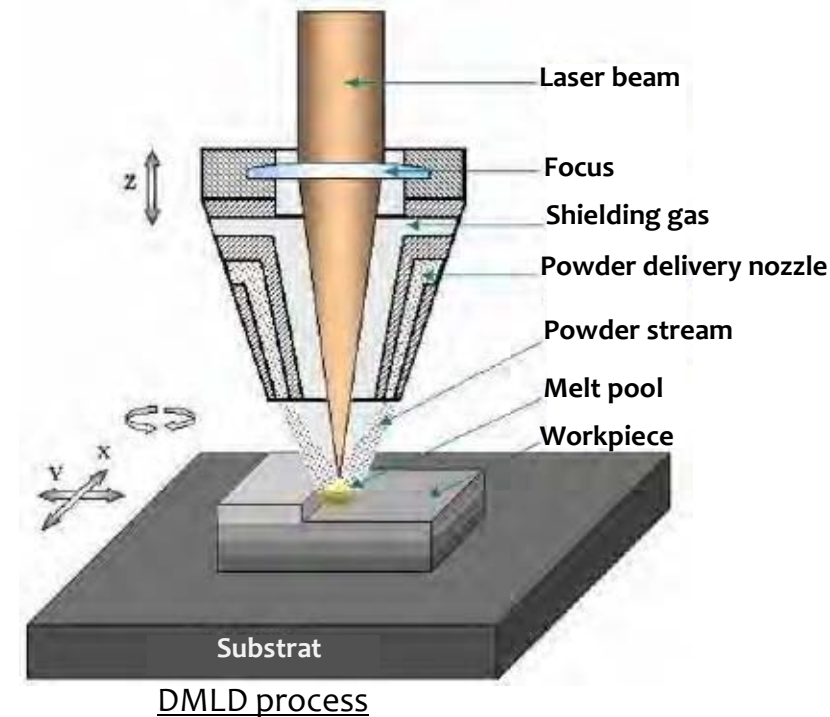
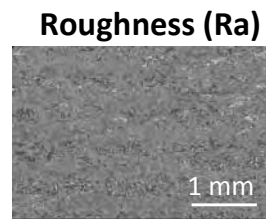
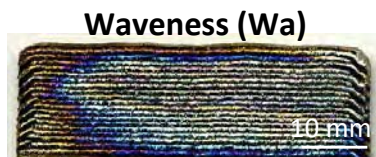
Three operating parameters :

- laser power (W)
- powder mass flux ( $\text{kg}\cdot\text{s}^{-1}$ )
- travel speed ( $\text{m}\cdot\text{s}^{-1}$ )



Main current limitations of DMLD processes : **surface finish**

Two surface finish criteria :



Project goals :

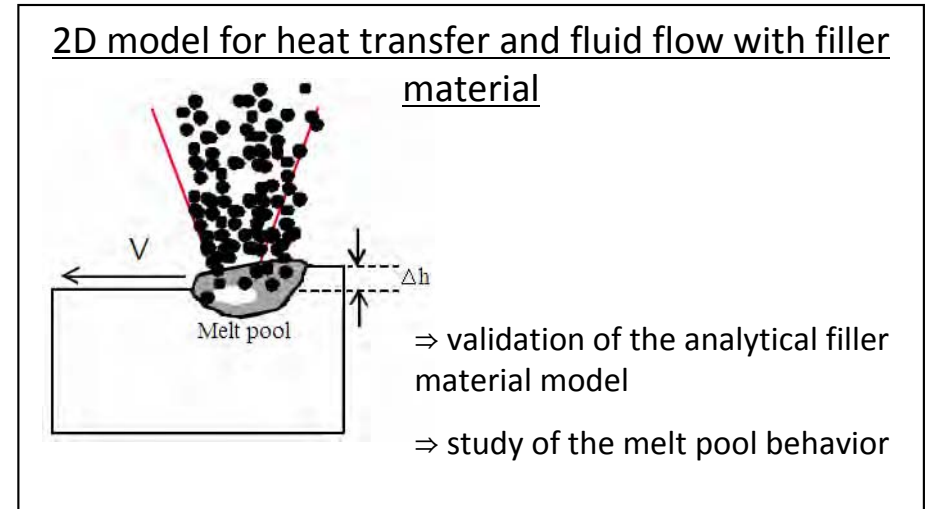
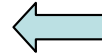
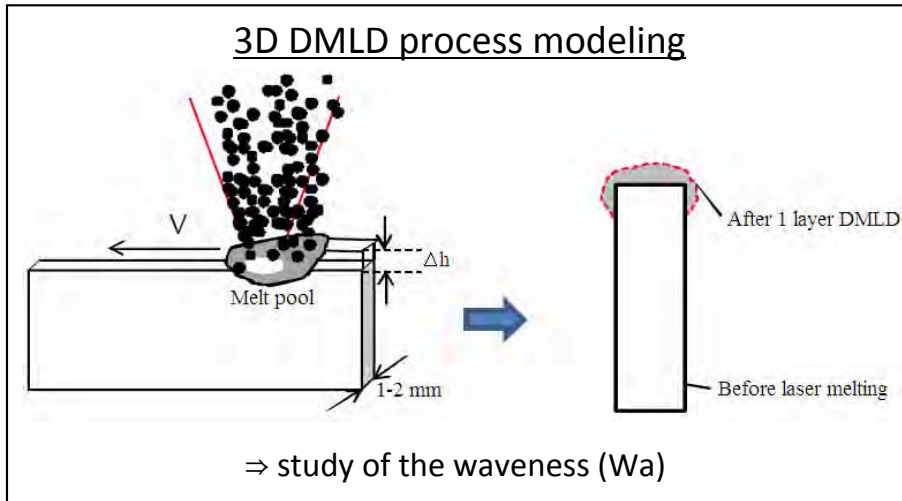
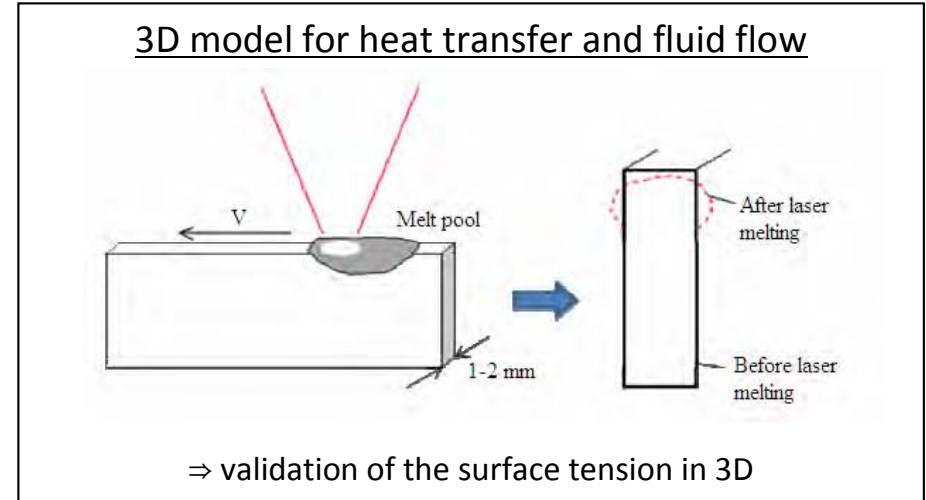
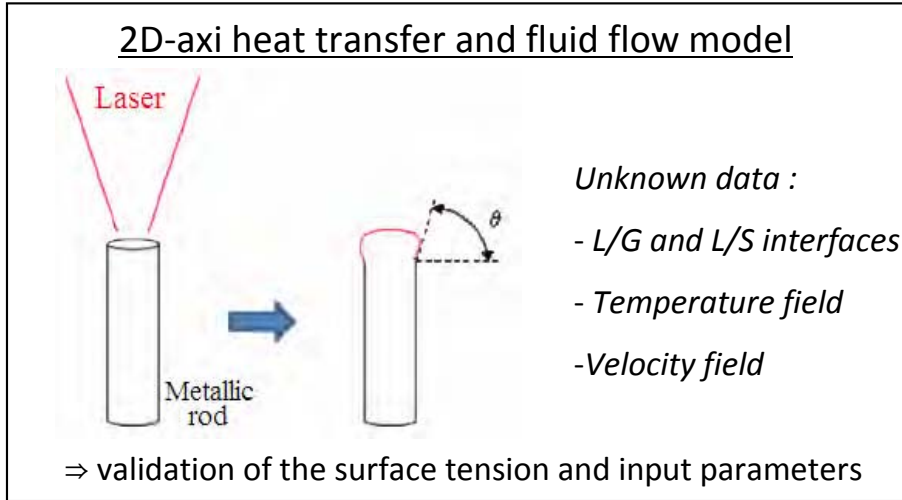
- Provide a real physical understanding of the melt pool behaviour in DMLD
- Develop a predictive model of DMLD process



Improve surface finish to obtain surface state near surface machining

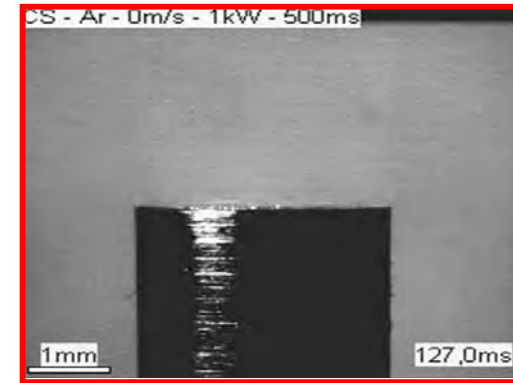
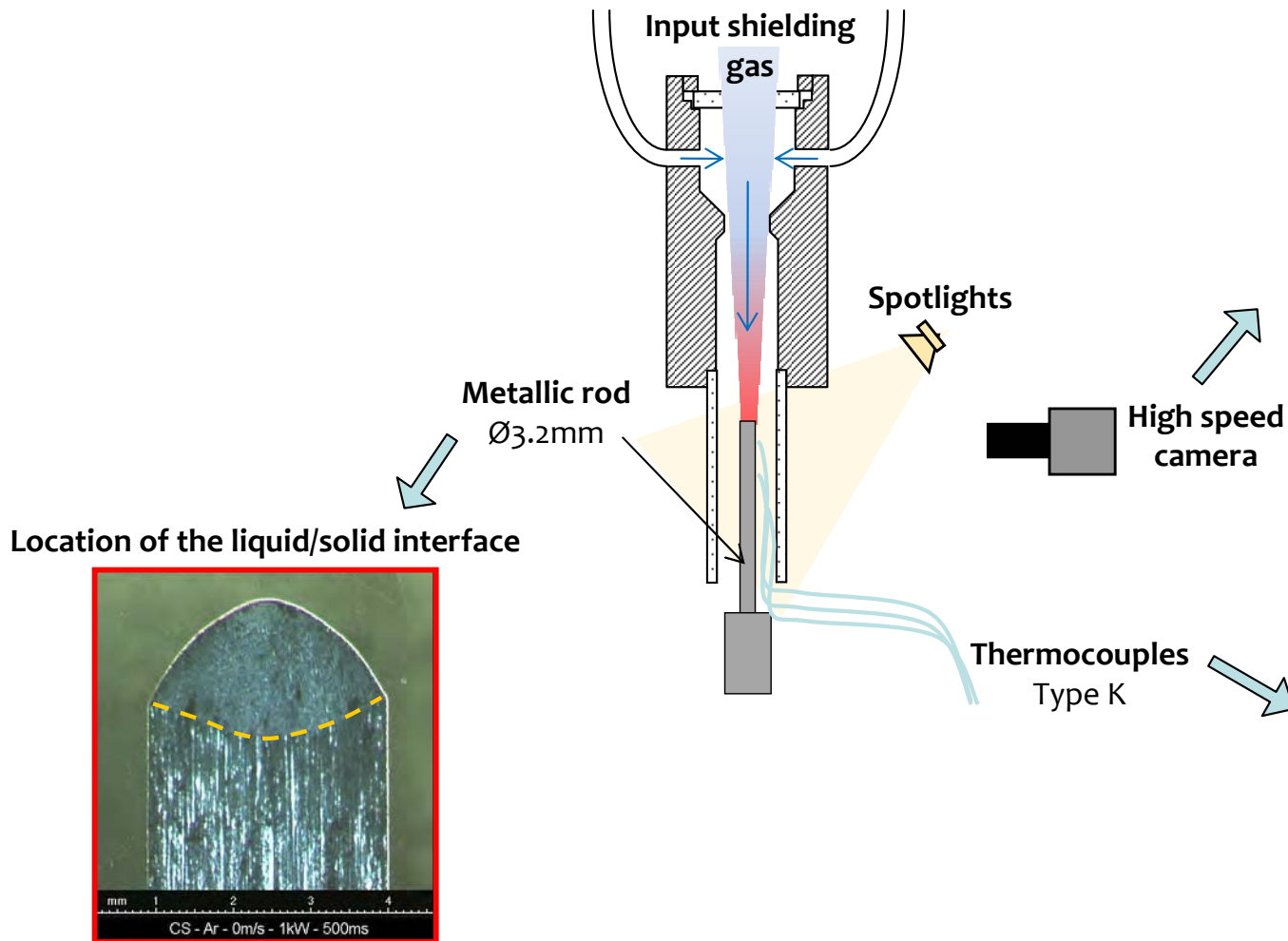
# Proposed approach

Laboratoire d'Ingénierie des MATériaux de Bretagne

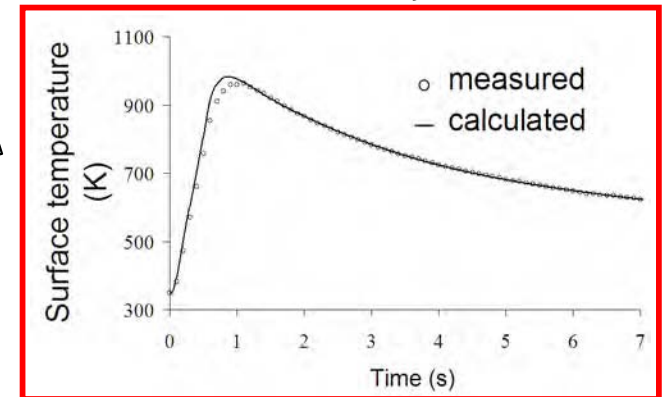


## Experimental set-up :

## Dynamic shape of the melt zone

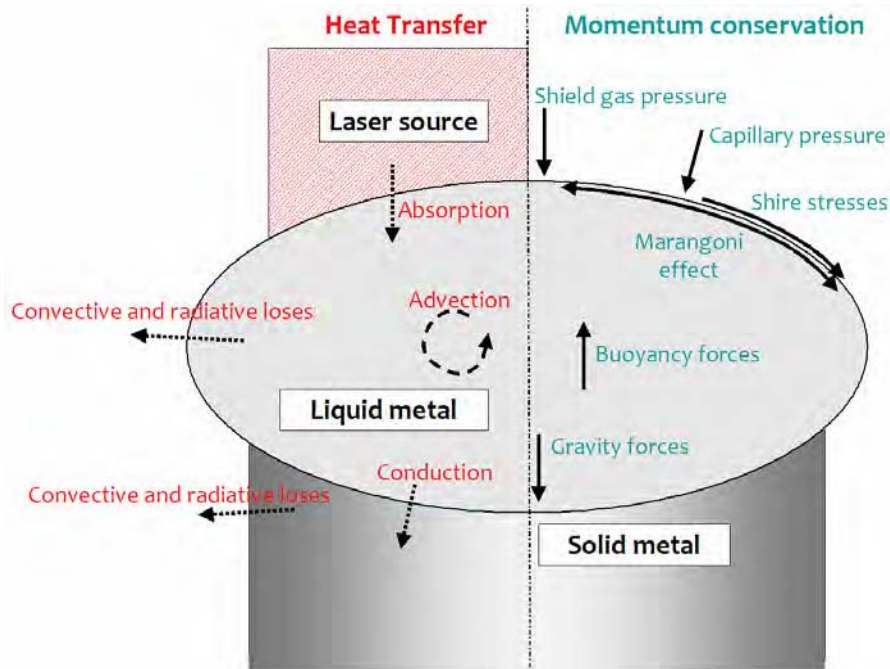


## Thermal cycles

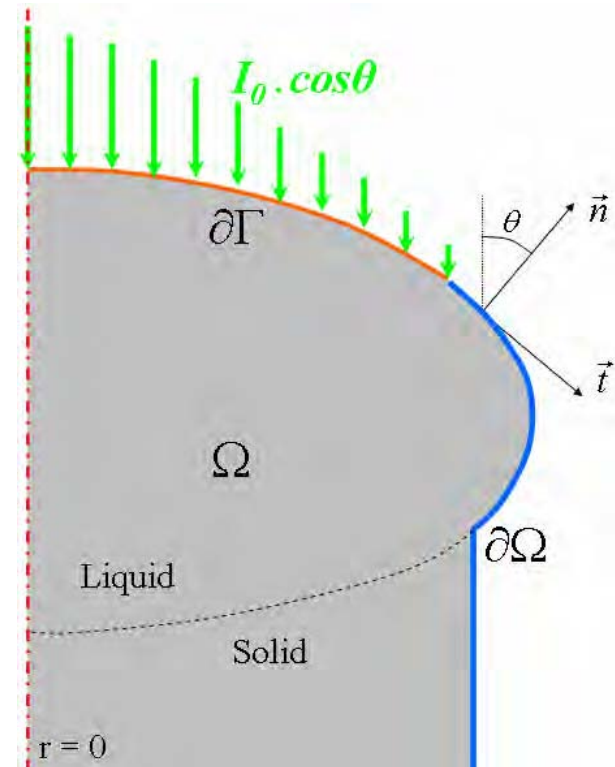


# 2D axial symmetry

- The model needs to describe several phenomena...



...coupled to the evolution of the laser/melt pool interaction and free surface deformation.



2Daxi model :

- fluid flow  $\rightarrow$  NS
- heat transfer  $\rightarrow$  HT
- moving mesh  $\rightarrow$  ALE

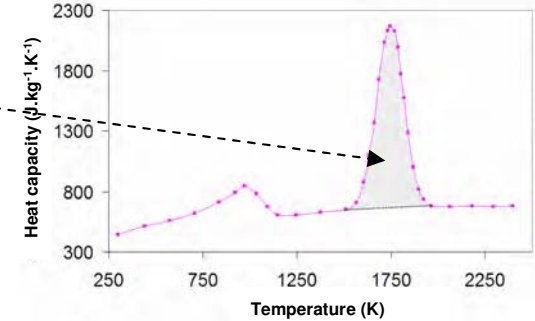
## Heat transfer equation

$$\rho(T) c_p^{app}(T) \left[ \frac{\partial T}{\partial t} + \vec{\nabla} \cdot (\vec{u}T) \right] - \vec{\nabla} \cdot (\lambda(T) \vec{\nabla} T) = 0$$

- equivalent  $c_p$  method

$$c_p^{app}(T) = c_p(T) + \Delta H_f \frac{df_L}{dT}$$

latent heat



## Momentum conservation equation

$$\rho_0 \left[ \frac{\partial \vec{u}}{\partial t} + \vec{u} \cdot (\nabla \vec{u}) \right] = \vec{\nabla} \cdot \left[ -pI + \mu(T) (\vec{\nabla} \vec{u} + (\vec{\nabla} \vec{u})^T) \right] + \vec{S}_s + \vec{F}_v$$

(incompressible Newtonian fluid)

- Darcy condition (liquid/solid interface)

$$\vec{S}_s = -C \frac{(1-f_L)^2}{(f_L^3 + b)} \vec{u}$$

$$f_L = \begin{cases} 0 & T \leq T_s \\ \frac{T - T_s}{T_L - T_s} & T_s < T \leq T_L \\ 1 & T > T_L \end{cases}$$

## Continuity equation

$$\vec{\nabla} \cdot \vec{u} = 0$$

## Moving mesh

→ ALE method (Winslow smoothing method)

- Volume forces (Buoyancy, gravity)

$$\vec{F}_v = \rho_0 \cdot (1 - \beta \cdot (T - T_0)) \cdot \vec{g}$$

- Mesh element size

- Fluid flow conditions

Surface tension :  $\sigma_n \vec{n} = -P_a \vec{n} + \gamma(T) \cdot \kappa \vec{n} \quad (\partial\Omega^1)$

Marangoni :  $\sigma_t = \frac{\partial\gamma}{\partial T}(T, S\%) \vec{\nabla}T \cdot \vec{i} \quad (\partial\Omega^1)$

- Moving mesh condition

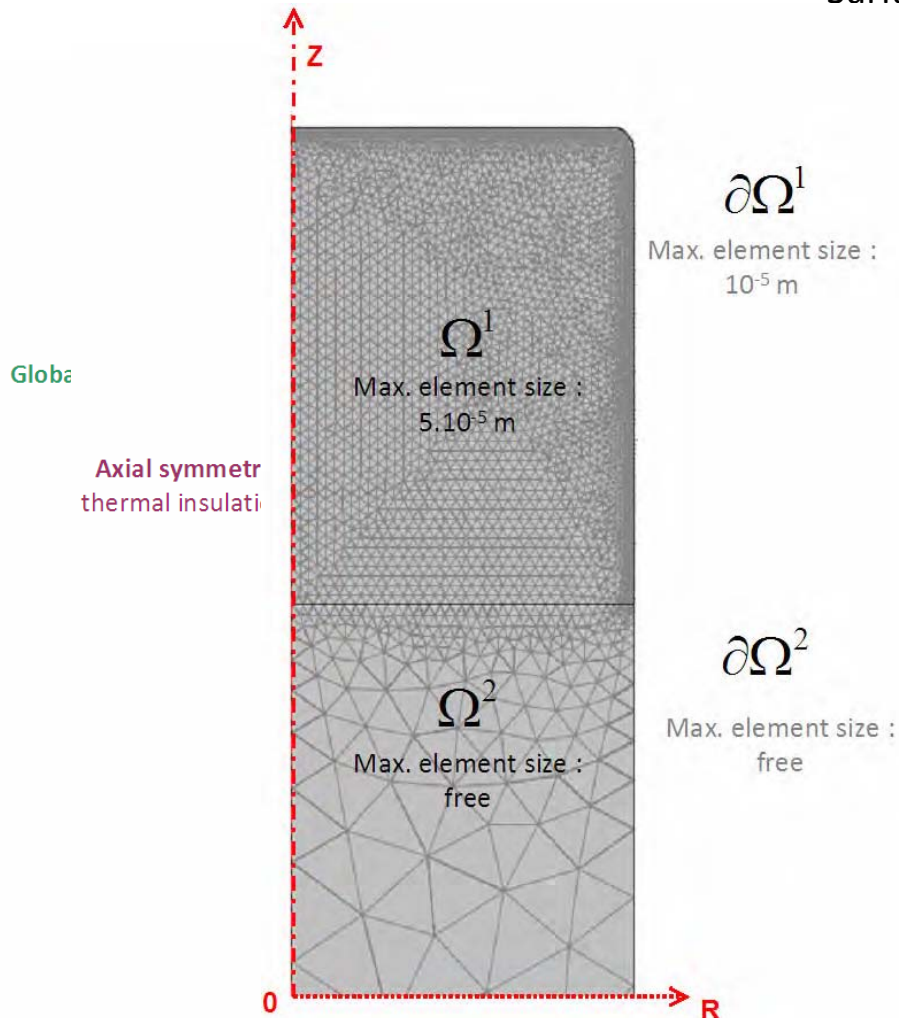
$$U_{mesh} \cdot \vec{n} = U_{material} \cdot \vec{n} \quad (\partial\Omega^1)$$

- Heat transfer conditions

$$q_{imp} = \begin{cases} \alpha(\theta) I_0(r, t) - h_c(T - T_0) - \epsilon\sigma(T^4 - T_0^4) & (\partial\Omega^1) \\ -h_c(T - T_0) - \epsilon\sigma(T^4 - T_0^4) & (\partial\Omega^2) \end{cases}$$

with :  $I_0(r, t) = \begin{cases} \frac{P_l}{\pi r_l^2} \delta(t) & r \leq r_l \\ 0 & r > r_l \end{cases}$

$$\alpha(\theta) = \alpha_0 \cdot \cos(\theta)$$



## Input parameters :

Material

Properties for liquid phase ?

$$\left( \rho, c_p, \lambda, \mu, \gamma, \frac{\partial \gamma}{\partial T}, \varepsilon \right)$$

Laser

Heat source

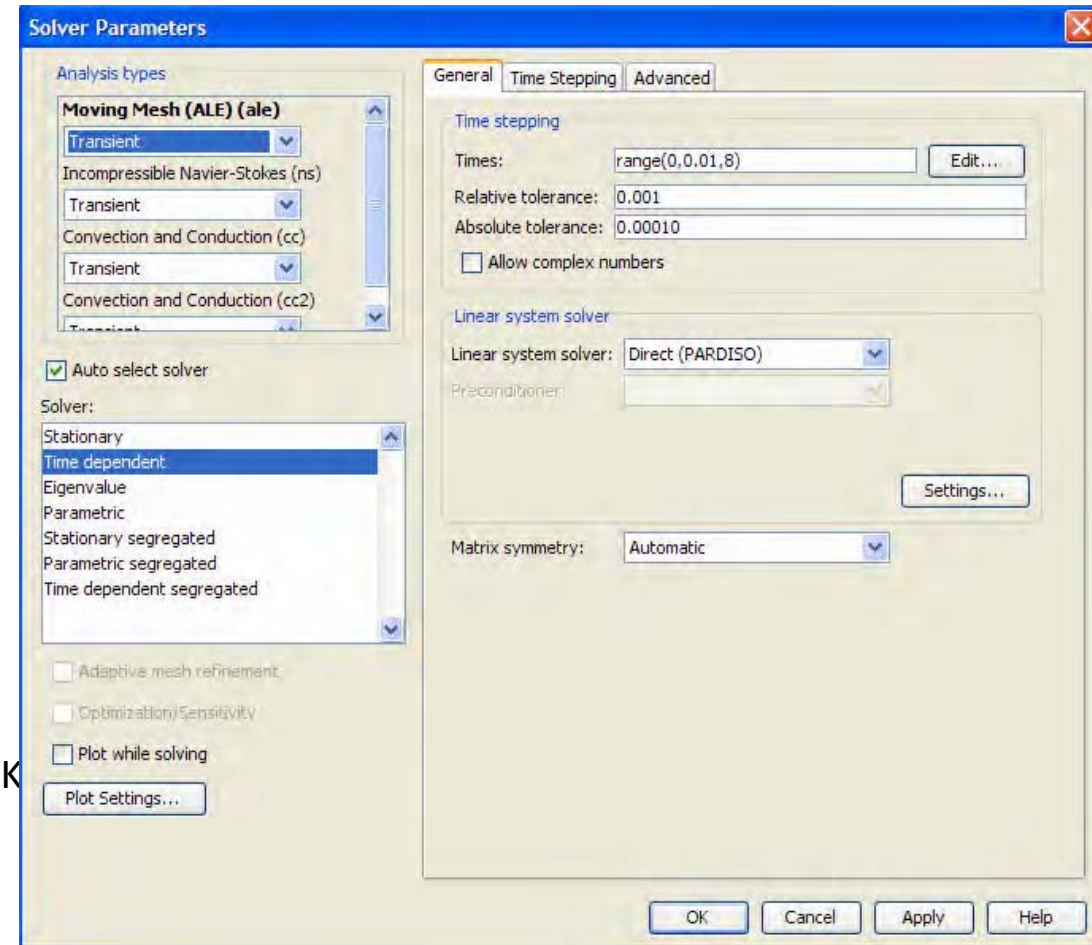
- Incident laser power : 962 W
- Laser beam radius : 1.57 mm
- Interaction time : 500 ms
- Absorptivity coefficient :  $\alpha_0$  ?

Heat losses

Convective and radiative loss

- Convective coefficient :  $15 \text{ W.m}^{-2}.\text{K}$
- Emissivity coefficient : 0.5

## Linear solver parameters



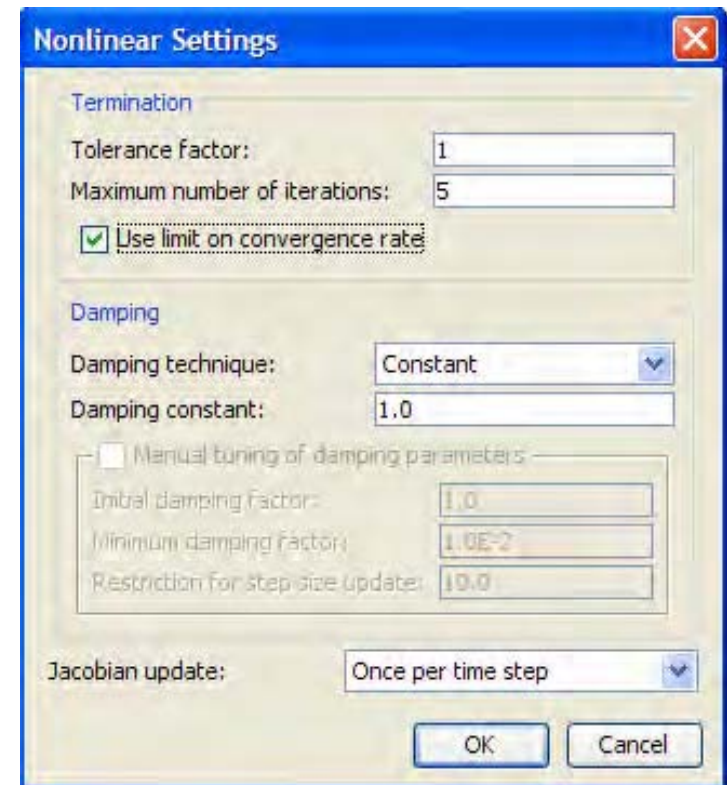
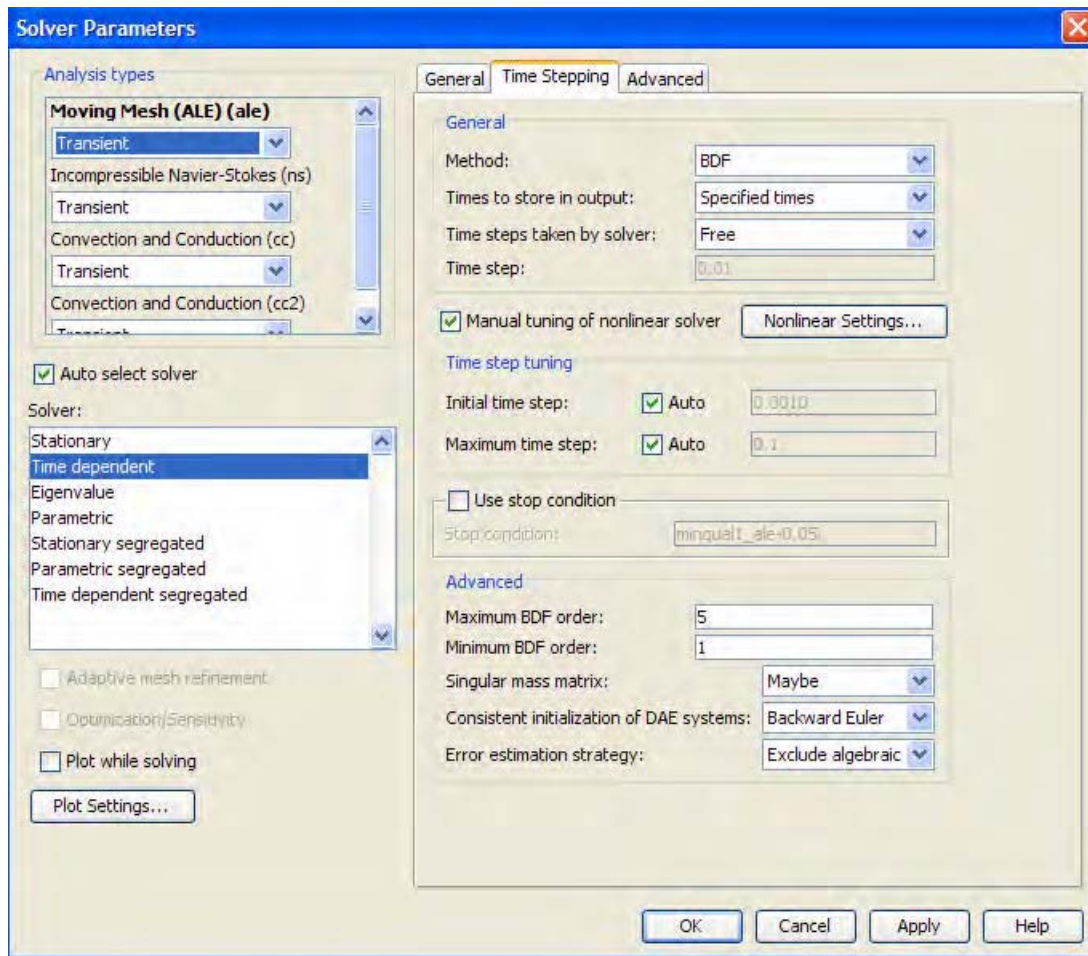


# Numerical results

## Input parameters :

### Non linear solver parameters

### Non linear parameters



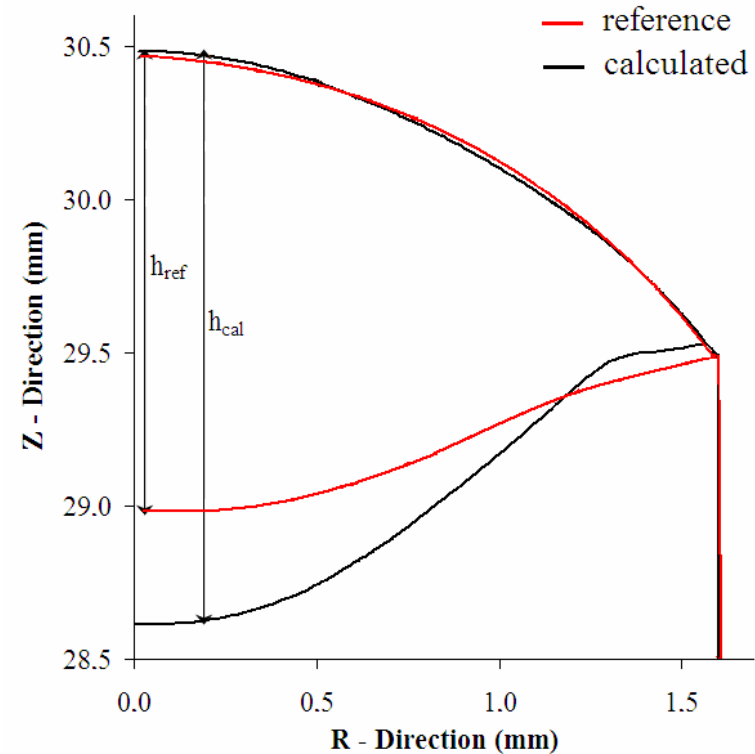
# Sensitivity analysis

	Reference values	Relative error (%) $(h_{cal}-h_{ref})/h_{ref}$
Thermal conductivity (S-L)	40-32 W.m <sup>-1</sup> .K <sup>-1</sup>	-13.5
Heat capacity (S-L)	500-710 J.kg <sup>-1</sup> .K <sup>-1</sup>	-23.8
Density (S-L)	7800-7290 kg.m <sup>-3</sup>	-22.1
Dynamic viscosity	5.10 <sup>-3</sup> Pa.s	-1.6
Capillary coefficient	1.5 N.m <sup>-1</sup>	<1
Thermocapillary coefficient	10 <sup>-4</sup> N.m <sup>-1</sup> .K <sup>-1</sup>	2.1
Absorptivity coefficient	0.3	32.4
Emissivity coefficient	0.5	<1
Latent heat of fusion	2.5.10 <sup>5</sup> J.kg <sup>-1</sup>	1

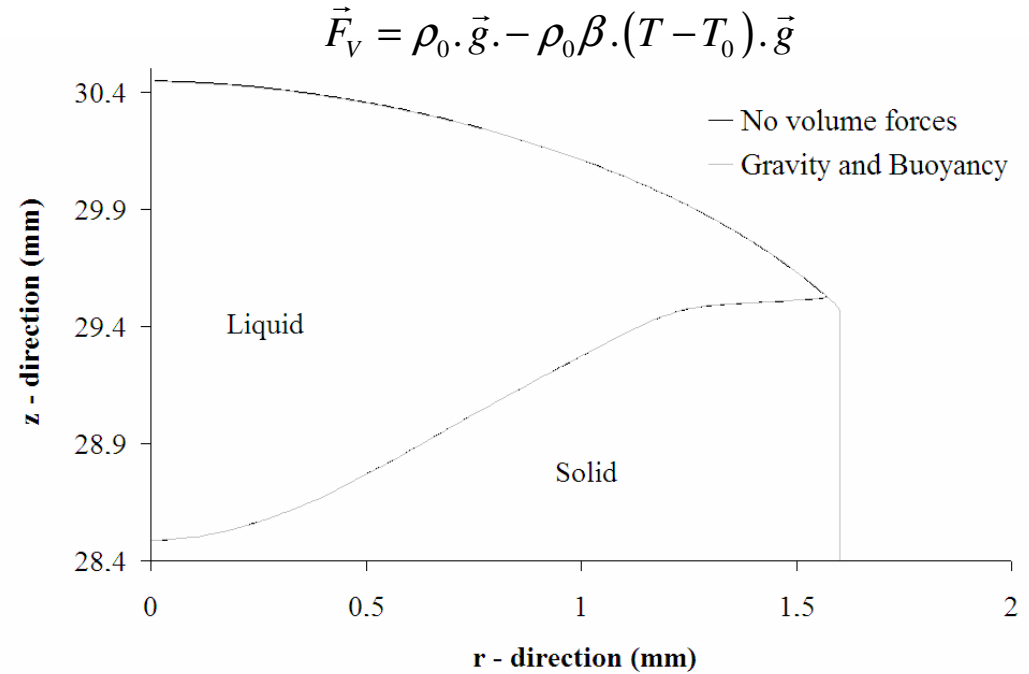
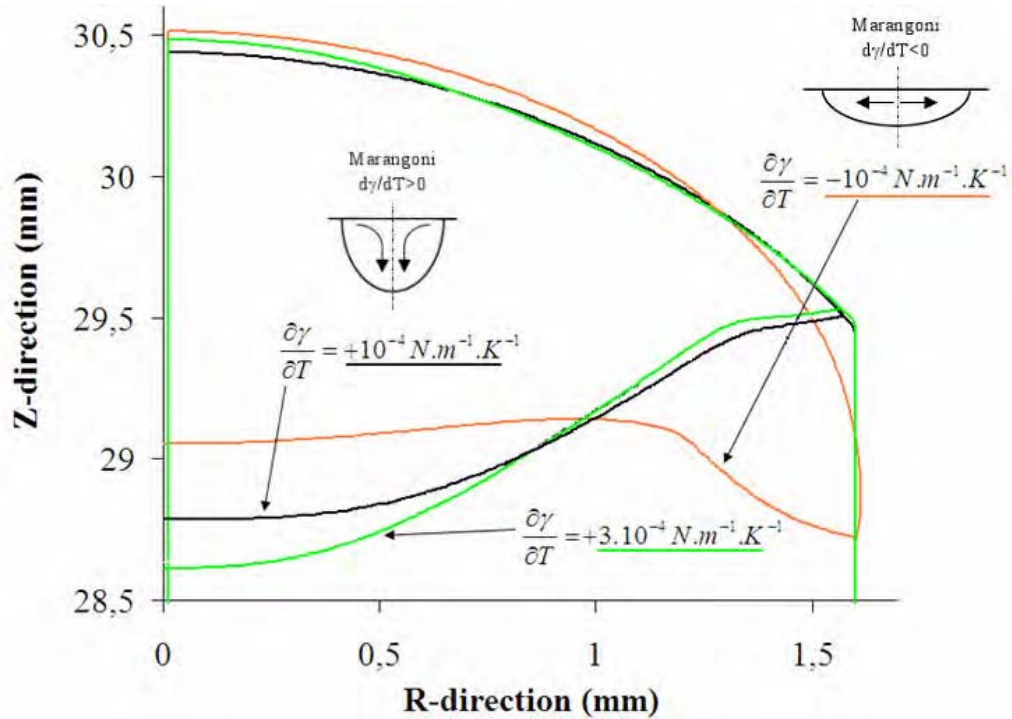
Each parameter is independently increased by 25% to evaluate his sensibility on the melt pool depth.

## Conclusions from sensitivity analysis :

- Input data very influent on melt pool geometry : thermal diffusivity and absorptivity
- Thermocapillary forces strongly control melt pool geometry
- Gravity and Buoyancy forces can be neglected

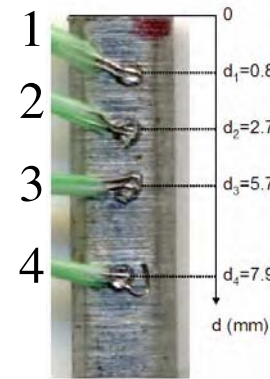
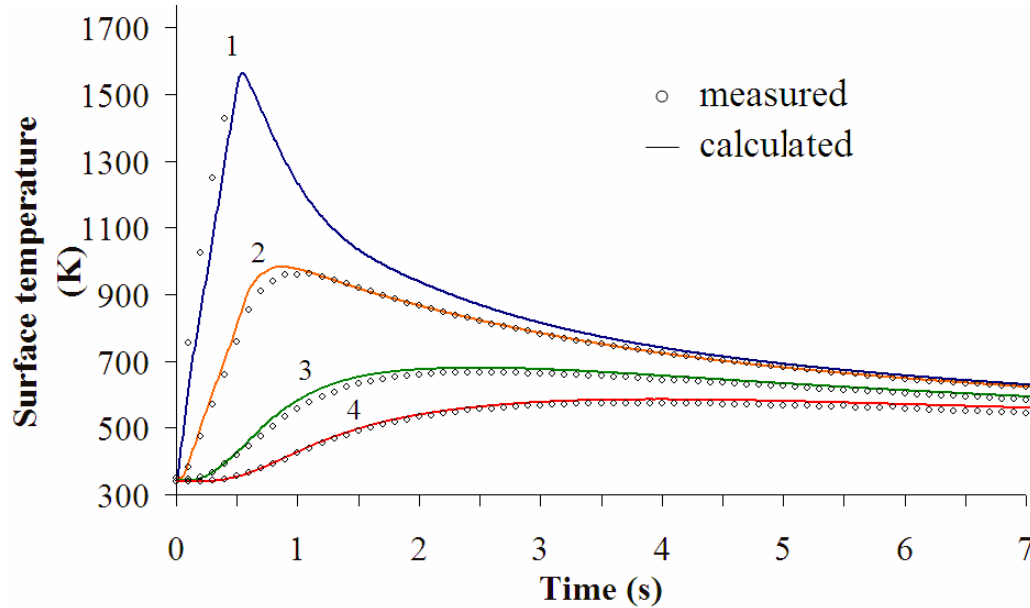


# Sensitivity analysis



## Conclusions from sensitivity analysis :

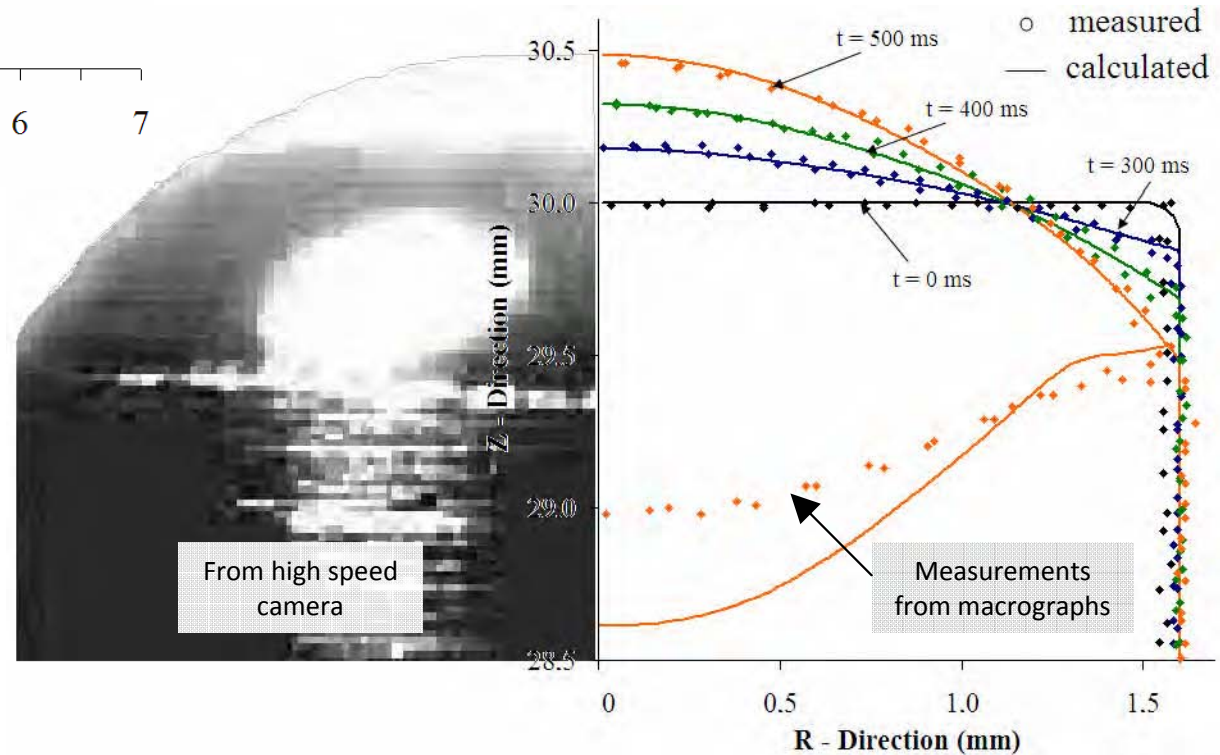
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## Conclusions :

Good correlation between numerical and experimental results for thermal cycles and liquid/gas interface location

Best fit for liquid/solid interface by adjusting  $\frac{\partial \gamma}{\partial T}$  as shown by the sensibility analysis



- 2D axial-symmetric model well describes physics thermohydraulic phenomena involved in local metallic rod melting :
  - Good correlation for :
    - » thermal cycles
    - » Dynamic shapes of liquid/gas interface
    - » Liquid/solid interface location
- Simplifying assumptions are validated :
  - » Gravity and Buoyancy forces negligible in our case
- Thermal properties, absorptivity coefficient and thermocapillary coefficient are key parameters for the prediction of the geometry
- Next steps :
  - » Validation of TA6V titanium alloy and 316L steel properties
  - » Implementation of a 2D thermohydraulic model with powder feeding
  - » Computation in a 3D framework for DMLD process modeling